where \underline{A}_c is defined in Claim 1 with $\gamma_c = \tau \rho$. Define To show contradiction, assume that $V_{\eta_k} \to \infty$ as $V_{\sigma} = \sigma^T P_c \sigma$ then, we get that the difference equation $\Delta V_{\sigma_k} = V_{\sigma_{k+1}} - V_{\sigma_k}$ along the trajectories of $\sigma(k+1) =$ $A_c\sigma(k)$ yields

$$\Delta V_{\sigma_k} = V_{\sigma_{k+1}} - V_{\sigma_k}$$

= $\sigma^T(k) \left[A_c^T P_c A_c - P_c \right] \sigma(k)$.

It is easy to see from Claim 1 with $\gamma_c = \tau \rho$, that

$$\Delta V_{\sigma_k} = -\tau \rho \sigma^T(k) P_c \sigma(k) -\tau \rho (1 - \tau \rho)^n \sigma^T(k) F^T F \sigma(k) \Delta V_{\sigma_k} \leq -\tau \rho \|\sigma(k)\|_P^2$$

Using this bound we now evaluate the difference equation $\Delta V_{\eta_k} = V_{\eta_{k-1}} - V_{\eta_k}$ where $V_{\eta_k} = \eta(k)^T P_c \eta(k)$ along the trajectories of (3.13) to obtain

$$\begin{array}{lcl} \Delta V_{\eta_{k}} & = & V_{\eta_{k-1}} - V_{\eta_{k}} \\ & \leq & -\tau\rho \left\| \eta(k) \right\|_{P_{c}}^{2} + \tau^{2}N^{2} \left\| \varepsilon(k) \right\|_{P_{c}}^{2} \\ & & + 2\tau N \left\| \varepsilon(k) \right\|_{P_{c}} \left\| \eta(k) \right\|_{P_{c}} \\ & \leq & -\tau \left(\rho - 1 \right) \left\| \eta(k) \right\|_{P_{c}}^{2} + \tau N^{2} \left\| \varepsilon(k) \right\|_{P_{c}}^{2} \end{array}$$

where we defined $N := \|\Omega_{\rho} \Delta_{\theta}^{-1} K C \Delta_{\theta}^{-1}\|$. Evaluating the sum from k_0 to ∞ on both sides of the inequality above, and using (3.12) we obtain that

$$\sum_{k=k_0}^{\infty} \Delta V_{\eta_k} \geq \tau \sum_{k=k_0}^{\infty} \left\{ (\rho-1) \|\eta(k)\|_{P_c}^2 - cN^2 \|\varepsilon(k)\|^2 \right\} \text{for displacement, respectively. The link inertia } I_l, \text{ the motor rotor inertia } J_m, \text{ the elastic constant } k, \text{ the link mass } m, \text{ the gravity constant } g, \text{ the center of mass } l \text{ and the viscous friction coefficients } F_l \text{ and } F_m \text{ are possibly properties}.$$

which implies that

$$\sum_{k=k_0}^{\infty} \left\| \eta(k) \right\|^2 \leq \frac{c}{\tau \left(\rho - 1\right)} \left(\left\| \eta(k_0) \right\|^2 + \frac{N^2}{\delta} \left\| \varepsilon(k_0) \right\|^2 \right)$$

hence, setting $\rho_{\min} > 1$ and since $\eta = \Omega_{\rho} z$, we finally obtain that

$$\left(\sum_{k=k_0}^{\infty} \|z(k)\|^2\right)^{1/2} \le \frac{c}{\sqrt{\tau}} \|\xi(k_0)\| \tag{3.14}$$

where it is clear that c is independent of τ . To determine the last bound, we recall that

$$\Delta V_{\eta_k} \le -\tau \left(\rho - 1\right) V_{\eta_k} + \tau N^2 \left\| \varepsilon(k) \right\|_{P_c}^2.$$

Then, using $\|\varepsilon(k)\|_{P_o} \le \|\varepsilon(k_0)\|_{P_o} e^{-\delta \tau (k-k_0)}$, we ob-

$$\Delta V_{\eta_k} \le -\tau \left(\rho - 1\right) V_{\eta_k} + \tau c N^2 \left\| \varepsilon(k_0) \right\|^2 e^{-2\delta \tau (k - k_0)}.$$

 $k \to \infty$. From the above we see that there exists $k^* > 0$, such that $\Delta V_{\eta_k} \leq 0$, which implies that $\|\eta(k)\|_{0}^{2} \leq c \|\eta(k_{0})\|_{0}^{2}$ for all $k \geq k^{*}$. On the other hand, $\|\eta(k)\|^2 \le c \|\eta(k_0)\|^2 + k^* N \tau_{\max} \|\varepsilon(k_0)\|^2$ for all $k < k^*$. Therefore, $\|\eta(k)\|^2 \le c \|\eta(k_0)\|^2 + ck^*N\boldsymbol{\tau}_{\max}\|\varepsilon(k_0)\|^2$ for all $k \geq k_0$. We conclude that there exists c > 0independent of τ such that

$$||z(k)|| \le c||\xi(k_0)|| \quad \forall k \ge 0.$$
 (3.15)

From the bounds (3.11), (3.12), (3.14), (3.15), and invoking Lemma 3 with $\nu = c$, p = 2 and $c_{\tau} :=$ $c\left(\max\{\frac{1}{\tau\delta},\frac{1}{\tau}\}\right)^{1/2}$ (which is obviously proportional to $\tau^{-1/2}$), we conclude that there exist $\kappa>0$ and λ_{τ} , proportional to τ , such that (2.2) holds.

4. Application to a flexible-joint robot

We apply the results developed above to the control of the flexible-joint robot. The dynamic equations of a single link robot arm with a revolute elastic joint rotating in a vertical plane are given by

$$J_l\ddot{q}_1 + F_l\dot{q}_1 + k(q_1 - q_2) + mgl\sin(q_1) = 0$$

$$J_m\ddot{q}_2 + F_m\dot{q}_2 - k(q_1 - q_2) = u$$

$$y = q_1$$

in which q_1 and q_2 are the link displacement and the romotor rotor inertia J_m , the elastic constant k, the link mass m, the gravity constant g, the center of mass land the viscous friction coefficients F_l and F_m are positive constant parameters. The control u is the torque delivered by the motor. Assuming that only q_1 is measured, u is to be designed so that q_1 tracks a desired reference $q_{r1}(t)$ where the parameters are assumed to be known. Defining the state variables,

$$\xi_1 = q_1, \quad \xi_2 = \dot{q}_1, \quad \xi_3 = q_2, \quad \xi_4 = \dot{q}_2,$$

the model in state-space form is

$$\dot{\xi}_{1} = \xi_{2}
\dot{\xi}_{2} = -\frac{F_{l}}{J_{l}} \xi_{2} - \frac{mgl}{J_{l}} \sin(\xi_{1}) - \frac{k}{J_{l}} (\xi_{1} - \xi_{3})
\dot{\xi}_{3} = \xi_{4}
\dot{\xi}_{4} = -\frac{F_{m}}{J_{m}} \xi_{4} - \frac{k}{J_{m}} (\xi_{1} - \xi_{3}) + \frac{1}{J_{m}} u.$$
(4.1)

4.1. Control design

The system (4.1) is state-feedback linearizable by means of the change of coordinates (cf. [5])

$$x_1 = \xi_1,$$